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## DETERMINATION OF TORQUE AND VIBRATION AMPLITUDE OF ROLLER MECHANISMS WITH BEARING SUPPORT WITH BELT CONVEYOR

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*Abstract. The article presents ideas about the reliability and durability of belt conveyors in mining enterprises today, as well as the constantly increasing requirements for equipment. The recommended values are presented through graphs representing the influence of the change in the vibration amplitude of the belt conveyor roller mechanism on the bearing flange and the roller mechanism shell, as well as the dependence of the torque on the axis of the roller mechanism on the change in the belt conveyor performance. As a result, the analysis of scientific studies about the possibility of increasing the UWK compared to the traditional one has been presented.*

*Key words. Conveyor, roller mechanism, belt element, deformation, loading, transportation, amplitude, vibration, technology.*

### Introduction

Today, large-scale scientific and research work is being carried out to improve resource-saving techniques and technologies of mining enterprises and to create their improved constructions. In this regard, including increasing the efficiency and productivity of belt conveyors, which are considered the main transport vehicles of mining enterprises, equipping their components with resource-efficient structures, increasing their durability, improving the operational reliability of the equipment, ensuring the process, developing and optimizing mathematical models. transportation of minerals using these methods is gaining importance. At the same time, it is important to develop the construction of roller mechanisms with a bearing support of a resource-efficient belt conveyor of a new design, justify its parameters, and reduce energy consumption.

### Literature review

In a series of experimental and theoretical researches, it is aimed to calculate the dynamic loads on the belt conveyor belts that work at high loads in mining enterprises, i.e., transporting large lumps, as a result, two approaches to eliminate the causes of dynamic loads have been formed. In the first approach, external shocks are considered as the main mechanism of impact of large load parts on the rollers due to the fact that the direction of the particle velocity vector and the tangent do not coincide with the roller surface at the point where the load vector is laid on the guide roller mechanisms

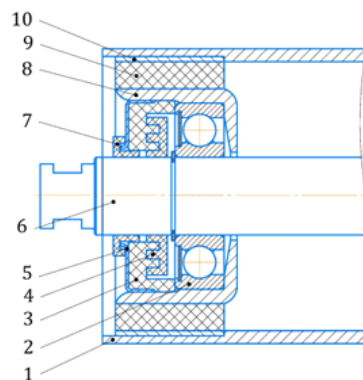
[1].

In addition to vibration contact repair of bearings installed in belt conveyor roller mechanisms, abrasive and friction repair types are also characteristic. This type of surface wear is predominant for the bearing supports of roller mechanisms. These types of wear determine the technical source of the bearings. The cause of this phenomenon is considered to be low-quality production of protective covers and sealing elements of the roller mechanism. However, belt conveyor roller mechanisms are such a mass product that perhaps the cost savings on protective cover elements justify themselves, so a type of wear not typical of bearings should be considered [2].

One of the popular studies in the field of determining the resistance to rotation of belt conveyor roller mechanisms is to determine the type of rotation resistance of roller mechanisms; to the quality and quantity of lubricants; to the rotation speed of the rollers; to the radial load on the roller; it was determined that it depends on the air temperature. After increasing the temperature of the bearing parts of the roller mechanisms, it was found that the power consumption and resistance to movement decreased. To take into account the effect of ambient temperature, an average value coefficient was proposed, the value of which depends on the type of lubrication and rotation speed [3].

### Discussion

Based on the study of the structural parameters of the belt conveyor roller mechanisms, the construction schemes of the belt-supported roller mechanisms were developed (Fig. 1).



1-shell, 2-bearing, 3, 4, 7-labyrinth cover, 5-protective washer, 6-axle, 8-bearing shell, 9-belt element bushing, 10-outer ring

**Figure 1. Construction of roller mechanism with elastic element bearing support (Belt conveyor)**

During the technological process, roller mechanisms are affected by the following forces: driving torque, gravity, inertia of unbalanced masses, frictional forces, technological pressures, etc. The forces forming the resultant force are directed radially and axially. These forces affect the bearing 2 through the bearing 2 and belt bushings 9 and 10 [4, 5, 6].

In this case, the presence of 9 belt bushings significantly reduces the effect of these forces on the bearing 2. In addition, due to the radial component of forces, the bending of axis 9 is significantly reduced. 9- the outer and inner rings of the belt bushing are made in the form of cones with the diameters of the bases  $d$  and  $d_{cut}$ , which allows to extend the impact forces along the axis. The use of supports with belt elements of the axles reduces the effect of vibrations on the housings of the roller mechanism due to the vibration of the rotating axle. Therefore, the vibration characteristics of these roller mechanisms are significantly reduced.

Loads of roller mechanisms with a bearing support with a recommended belt element mainly depend on the laws of change of the mass of raw materials transported on the conveyor belt. It should be noted that the mass of raw materials transported on a belt conveyor has random patterns in time [7, 8, 9]:

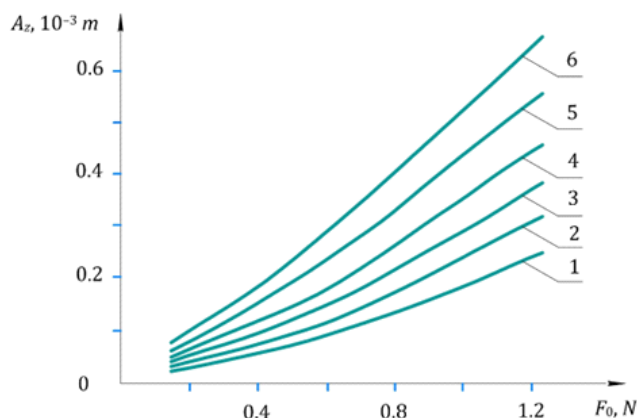
$$m_{rm} = m_1 + m_0 \sin \omega t \pm \delta m_0$$

where  $m_1$  – is the average mass of transported raw materials;  $m_0$  – is the amplitude of raw material mass change values;  $\omega$  – is the mass change frequency;  $\delta m_0$  – is a random generator.

The change in the mass of the raw material affects the bearing flange and main part support bushing. In this case, its friction force also changes, leading to feeding. In this case, it is important that the main rubber bushing-shock absorber with a strap element located between the outer flange of the composite bearing and the inner metal bushings is sufficiently absorbed from the loads. Therefore, the analysis of the vibrations of the bearing shell and the outer flange attached to it under the influence of external loading, and the justification of the parameters of the rubber bushing to reduce them are considered to be one of the main issues [10, 11].

## Results

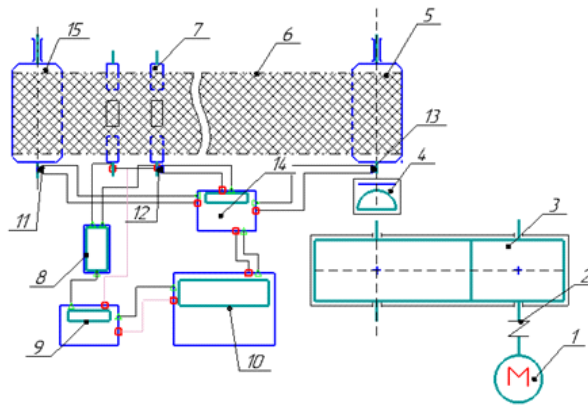
It can be seen from the resulting connection graphs in Figure 2 that when the load increases from 0.26 N to 1.9 N and the mass of the bearing shell is  $0.3 \cdot 10^2$  kg, the vibration amplitude of the roller mechanism is increases from  $0.4 \cdot 10^{-3}$  m to  $0.250 \cdot 10^{-3}$  m. If the mass of the bearing shell decreases to  $0.275 \cdot 10^2$  kg, the displacement amplitude of the roller mechanism reaches from  $0.5 \cdot 10^{-3}$  m to  $0.74 \cdot 10^{-3}$  m. Therefore, to ensure that the vibration amplitude of the roller mechanism does not exceed  $0.4 \cdot 10^{-3}$  m, it is advisable to make  $m_s \leq (0.275 \div 0.325) \cdot 10^2$  kg and  $F_0 \leq (1.02 \div 1.25)$  N.



**Figure 2. Dependence graphs of the influence of the change in the vibration amplitude of the roller mechanism on the bearing flange and the shell of the roller mechanism**

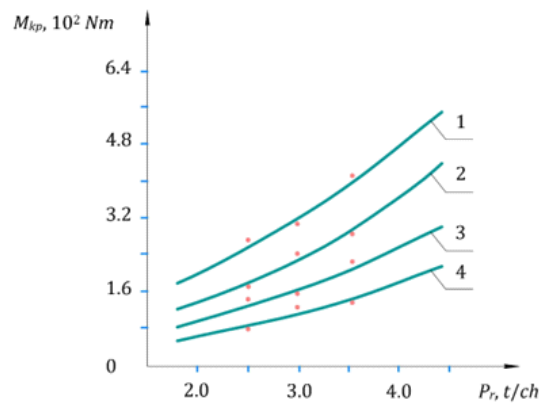
Studies have shown that an increase in the coefficient of uniformity of the bearing belt bushing not only decreases the vibration amplitude  $z$  and  $\dot{z}$ , but also causes an increase in the vibration frequency of the roller mechanism. When the force increases to 1.8 N, the value of  $A_z$  decreases linearly from  $0.59 \cdot 10^{-3}$  m to  $0.24 \cdot 10^{-3}$  m. Accordingly, the vibration amplitude of the speed of the roller mechanism also decreases nonlinearly, and the coefficient of uniformity of the belt element support of the roller mechanism bearing increases (see Fig. 2). The recommended values of the uniformity coefficient of the bearing belt support are  $c_1 = (5.0 \div 5.5) \cdot 10^4$  N/m;  $c_2 = (0.1 \div 0.12) \cdot 10^4$  N/m, where the vibration amplitude  $A_z = (0.09 \div 0.14) \cdot 10^{-3}$  m/s,  $A_{\dot{z}} = (0.08 \div 0.11) \cdot 10^{-3}$  m/s it is guaranteed to be between [12, 13].

Special sensors and the tensometric method were used to experimentally determine the torque on the shaft of the roller mechanism with a belt bushing, the frequency of rotation and the amplitude of vibration of the shaft. A special experimental copy of the belt conveyor was prepared and its parameters were measured based on the electrotensometric scheme. Based on experimental research, oscillograms and connection graphs were obtained [14, 15, 16].



1-engine; 2, 4-belt element coupling; 3-reducer; 5-leading drum; 6-strip; 7-roller; 8-UZB cable electronic thermocouple; 9, 14-vibXpert special number changer device (laboratory equipment); 10-computer, 11, 12, 13-strain gauges; 15-lead drum [17]

**Figure 3. Electrotensometric scheme of the experimental stand of the guide roller mechanism with a belt conveyor**



**Figure 4. Graphs representing the dependence of the torque on the axis of the roller mechanism on the change in the performance of the belt conveyor**

If the torque and vibration amplitude are reduced in the range of  $(50\div55) Nm$  when using a strap support, the torque is reduced by  $(35\div40)\%$  on average. (Figure 4). The performance increases from  $3.51 \cdot 10^2 Nm$  to  $6.23 \cdot 10^2 Nm$ , when using 7B-54MBC rubber, the torque increases from  $3.82 \cdot 10^2 Nm$  to  $4.1 \cdot 10^2 Nm$  in roller mechanism, these results are, when 7IRP13-46 rubber is used. increases to  $1.22 \cdot 10^2 Nm$ . This means that in order to reduce the torque in the roller mechanism, it is possible to use a rubber brand with a small rotational speed for the rubber bushing of the bearing, in which case it is advisable to choose the 7IRP13-48 brand rubber as the optimal option.

## Conclusion

As a result of the analysis of the operation process of belt conveyor roller mechanisms, it was found that the bearing support, which is the main working organ of the roller mechanism, quickly fails during operation, which has a negative effect on the conveyor technology. Due to the vibration of the roller mechanism, the support bearings quickly fail. The use of bearing supports with belt elements provides significant absorption of vibrations in the roller mechanism and increases the service life of the roller mechanism and bearings.

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